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AMENDMENTS TO THE SPECIFICATION:

Page 1, please add the following <u>new paragraphs</u> before paragraph [0001]:

[0000.2] CROSS-REFERENCE TO RELATED APPLICATIONS

[0000.4] This application is a 35 USC 371 application of PCT/DE 2004/002081 filed on September 17, 2004.

[0000.6] BACKGROUND OF THE INVENTION

Please replace paragraph [0001] with the following amended paragraph:

[0001] Prior Art Field of the Invention

Please add the following <u>new</u> paragraph after paragraph [0002]:

[0002.5] Description of the Prior Art

Please replace paragraph [0003] with the following amended paragraph:

[0003] Such valves Valves of the type described above are known from European Patent Disclosure EP 0 192 241. The injection valves, particular in common rail injection systems, are provided with servo valves for controlling the fluid flows. For supplying fuel to internal combustion engines, so-called storage injection systems are used, which work with very high injection pressures. Such injection systems are known as common rail systems for diesel engines and HPDI injection systems for Otto engines. In these injection systems, the fuel is pumped by a high-pressure pump into a common pressure reservoir, from which fuel is supplied to the injection valves at the individual cylinders. As a rule, the opening and closing of the injection valves are controlled electronically.

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Page 2, please replace paragraph [0006] with the following amended paragraph:

[0006] It is the **primary** object of the invention to create a valve of the type defined at the

outset that prevents the aforementioned unfavorable force ratios distribution and assures

secure sealing.

Please replace paragraph [0007] with the following amended paragraph:

[0007] Advantages of the Invention

SUMMARY AND ADVANTAGES OF THE INVENTION

Please replace paragraph [0008] with the following amended paragraph:

[0008] This above object is attained in that the actuator chamber has at least one additional

inlet bore, which enables a distribution of the introduction of force to the actuator cap and/or

to the ram.

Page 4, please replace paragraph [0016] with the following amended paragraph:

[0016] Drawing BRIEF DESCRIPTION OF THE DRAWINGS

Please replace paragraph [0017] with the following amended paragraph:

[0017] The invention is more fully described below in terms of an exemplary embodiment

shown in the drawings, herein below, in conjunction with the drawings, in which:

Please replace paragraph [0018] with the following amended paragraph:

[0018] Fig. 1, is a schematic sectional view along the longitudinal axis showing a detail of a

valve with a high-pressure inlet for a common rail injection system, in the prior art;

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Please replace paragraph [0019] with the following amended paragraph:

[0019] Fig. 2, **is** a schematic sectional view showing the high-pressure inlet of the valve in an embodiment according to the invention; **and**

Please replace paragraph [0020] with the following amended paragraph:

[0020] Fig. 3, **is** a schematic sectional view showing the high-pressure inlet of the valve in a further variant embodiment of the invention.

Page 5, please replace paragraph [0021] with the following amended paragraph:

[0021] Description of the Exemplary Embodiments

DESCRIPTION OF THE PREFERRED EMBODIMENTS

Page 7, please delete paragraph [0028].

Please replace paragraph [0029] with the following amended paragraph:

[0029] [[The]] Fig. 2 shows one of the preferred embodiments as an example of the invention wherein valve housing 10 has two diametrically opposed inlet bores 13, which discharge into the actuator chamber 11 in the region of the conical face 14, outside the annular sealing face 33 of the actuator cap 32. In this exemplary embodiment, the high-pressure inlet 12 is embodied centrally, along the center axis of the valve housing 10. As a result of this geometry, it can furthermore be achieved that the inlet bores 13 extend at an acute angle to the center axis of the valve housing 10. Moreover, as shown as an example in Fig. 2, a cross-sectional enlargement 18 may be provided, which is located between the high-pressure inlet 12 and the inlet bores 13. This cross-sectional enlargement 18 forms a high-

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pressure chamber and in the process reinforces a uniform distribution of the fuel quantity

among the individual inlet bores 13. The cross sections of the <u>two</u> inlet bores 13 may be

reduced compared to the cross section of an individual inlet bore 13.

Please replace paragraph [0030] with the following amended paragraph:

[0030] Fig. 3 shows a variant embodiment of the invention, in which the inlet bores 13,

compared to the variant shown in Fig. 2, discharge laterally into the actuator chamber 11.

Once again, the inlet bores 13 are located diametrically opposite one another, and hence a

more symmetrical introduction of force can result is attained.

Page 8, please add the following new paragraph after paragraph [0031]:

[0032] The foregoing relates to a preferred exemplary embodiment of the invention, it being

understood that other variants and embodiments thereof are possible within the spirit and

scope of the invention, the latter being defined by the appended claims.

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